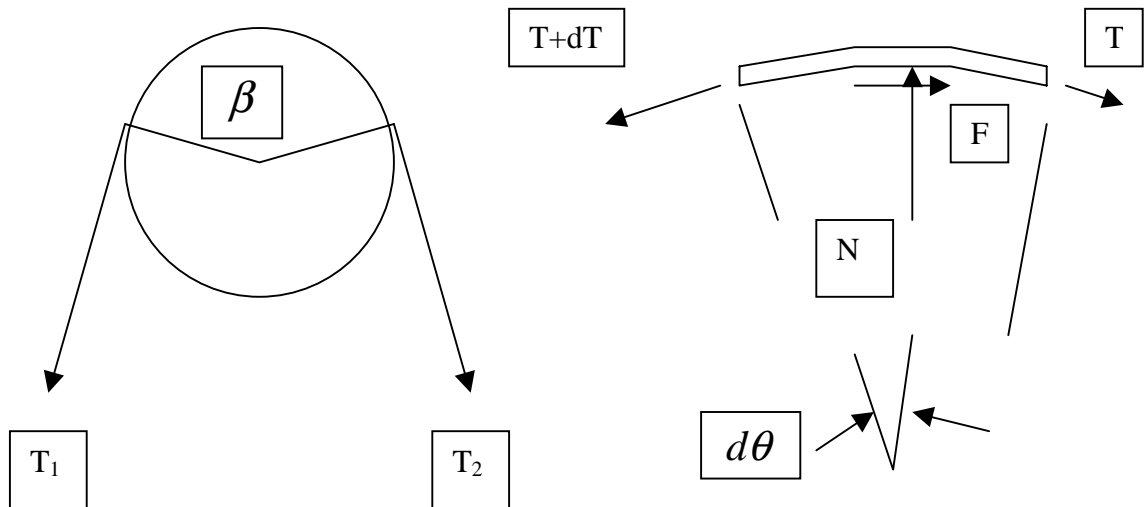


Design Notes - Belt Drives - Simple Analysis



Consider the equilibrium of a small element of belt subtending a small angle $d\theta$. Assume that the coefficient of friction between pulley and the belt is μ and that the belt tension on the tight side of the pulley is T_1 and slack side of the pulley T_2 . Assume the angle of wrap of the belt around the pulley is β and ignore centrifugal effects.

Resolving forces on the small element of belt in the tangential direction:

$$(T + dT) \cos \frac{d\theta}{2} - F - T \cos \frac{d\theta}{2} = 0 \text{ as } d\theta \text{ is small, } \cos d\theta \approx 1$$

so $dT = F$, when the belt is on the point of slipping: $F = \mu N$ so $dT = \mu N$

Resolving the forces on the small element of belt in a radial direction:

$$(T + dT) \sin \frac{d\theta}{2} + T \sin \frac{\theta}{2} - N = 0 \text{ for small angles } \sin \theta \approx \theta$$

so $Td\theta = N$

Eliminate N , which is not wanted at this stage, gives: $\frac{dT}{T} = \mu d\theta$

This can be integrated: $\int_{T_2}^{T_1} \frac{dT}{T} = \mu \int_0^\beta d\theta$ giving: $\ln \frac{T_1}{T_2} = \mu\beta$

$$\text{or: } \frac{T_1}{T_2} = e^{\mu\beta}$$